ENGINEERING


## Description

This report studies a Tee component and the correlation between stresses, SIFs at different branch angles due to inplane and out-of-plane bending moment.

The reason for the initialization of the report was discussion thread in the LinkedIN community and my own curiosity. I hope this report might shed some light on the discussion.

DISCLAIMER: The analysis, results and conclusion found in this report are based on a quick and shallow FEA and are ONLY for guidance and a SUPPLEMENT to the LinkedIN discussion.

## Simulation of Tees

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Study name: Design study Analysis type: Design Study, static

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## NOTE: The standard SolidWorks

Simulation standard report was chosen to ease the reporting time. Please note that this is not common practice at Stressman Engineering. For paid projects we are using more professional and in depth reports.

## Input and Assumptions

Header input
$\mathrm{OD}=114.3$ (4ND), $\mathrm{WT}=5 \mathrm{~mm}$
Length of header $=800 \mathrm{~mm}$.
Branch input
OD_Branch = From 54.3 mm to 114.3 mm in steps of 10 mm
Angle $=$ From 30 degrees to 90 degrees in steps of 10 degrees
WT $=5 \mathrm{~mm}$
Length of branch $=300 \mathrm{~mm}$
Crotch radius (rx) $=5 \mathrm{~mm}$

## Boundaries



Study 1 - In-plane bending of header


Study 1 - Out-of-plane bending of header


Study 2 - In-plane bending of branch


## Abbreviations

Deg = Degrees, FEA = Finite Element Analysis, LC = Load case, ND = Nominal diameter, Nom = Nominal stress, OD = Outer diameter, OP=Out-of-Plane, IP = In-plane, SIF = Strength intensification Factor, WT = Wall thickness.


## Units

| Unit system: | SI (MKS) |
| :--- | :--- |
| Length/Displacement | mm |
| Temperature | Kelvin |
| Angular velocity | Rad/sec |
| Pressure/Stress | $\mathrm{N} / \mathrm{mm}^{\wedge} 2(\mathrm{MPa})$ |

## Calculation of SIF

The ASME codes use girth welds as "base lines" /1/. This means that a SIF of 2.0 is already incorporated into the code and its safety factors. Therefore should the peak stress found in an FEA be divided by 2 times the nominal stress in a straight pipe.

$$
\boldsymbol{S I F}=\frac{\text { Peak stress }}{2 \text { Nominal stress in pipe }}
$$

## Design Study Setup

## Design Variables

| Name | Type | Values | Units |
| :--- | :--- | :--- | :--- |
| Angle | Range with Step | Min:30 Max: $90^{\circ}$ Step: $10^{\circ}$ | deg |
| Branch OD | Range with Step | Min:54.3 Max:114.3 Step:10 | mm |

## Monitors

| Sensor name | Condition | Study name | Comment |
| :--- | :--- | :--- | :--- |
| Stress1 | Monitor Only | Study 1-IP Header | In-plane moment applied on header |
| Stress2 | Monitor Only | Study 2-IP Branch | In-plane moment applied on branch |
| Stress1 | Monitor Only | Study 1-OP Header | Out-of-plane moment applied on header |
| Stress2 | Monitor Only | Study 2-OP Branch | Out-of-plane moment applied on branch |

## Study Results

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The table below shows the results from the studies.

| Angle [deg] | OD <br> Branch [mm] | In-plane stresses [Mpa] |  | In-plane SIFs$[-]$ |  | Out-of-plane stresses [Mpa] |  | $\begin{gathered} \text { In-plane SIFs } \\ {[-]} \end{gathered}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Stress 1 | Stress 2 | SIF 1 | SIF 2 | Stress 1 | Stress 2 | SIF 1 | SIF 2 |
| Nom | 54.3 | 22.40 | 114.22 | 0.50 | 0.50 | 22.40 | 114.22 | 0.50 | 0.50 |
| 30 | 54.3 | 60.06 | 315.85 | 1.34 | 1.38 | 22.55 | 263.63 | 0.50 | 1.15 |
| 40 | 54.3 | 48.08 | 298.31 | 1.07 | 1.31 | 24.68 | 396.93 | 0.55 | 1.74 |
| 50 | 54.3 | 53.54 | 267.78 | 1.20 | 1.17 | 23.53 | 556.69 | 0.53 | 2.44 |
| 60 | 54.3 | 57.74 | 306.54 | 1.29 | 1.34 | 23.29 | 700.35 | 0.52 | 3.07 |
| 70 | 54.3 | 59.36 | 334.54 | 1.32 | 1.46 | 22.94 | 784.42 | 0.51 | 3.43 |
| 80 | 54.3 | 61.43 | 342.16 | 1.37 | 1.50 | 23.34 | 848.39 | 0.52 | 3.71 |
| 90 | 54.3 | 61.89 | 351.60 | 1.38 | 1.54 | 22.40 | 844.31 | 0.50 | 3.70 |
| Nom | 64.3 | 22.40 | 77.97 | 0.50 | 0.50 | 22.40 | 77.97 | 0.50 | 0.50 |
| 30 | 64.3 | 62.18 | 220.31 | 1.39 | 1.41 | 23.93 | 207.17 | 0.53 | 1.33 |
| 40 | 64.3 | 51.81 | 193.05 | 1.16 | 1.24 | 22.71 | 327.50 | 0.51 | 2.10 |
| 50 | 64.3 | 59.04 | 189.00 | 1.32 | 1.21 | 22.62 | 439.40 | 0.51 | 2.82 |
| 60 | 64.3 | 63.06 | 222.66 | 1.41 | 1.43 | 23.21 | 545.88 | 0.52 | 3.50 |
| 70 | 64.3 | 65.60 | 242.46 | 1.46 | 1.55 | 23.32 | 632.17 | 0.52 | 4.05 |
| 80 | 64.3 | 67.51 | 249.03 | 1.51 | 1.60 | 23.33 | 665.32 | 0.52 | 4.27 |
| 90 | 64.3 | 68.12 | 251.62 | 1.52 | 1.61 | 23.20 | 703.22 | 0.52 | 4.51 |
| Nom | 74.3 | 22.40 | 56.56 | 0.50 | 0.50 | 22.40 | 56.56 | 0.50 | 0.50 |
| 30 | 74.3 | 65.71 | 173.71 | 1.47 | 1.54 | 22.75 | 169.64 | 0.51 | 1.50 |
| 40 | 74.3 | 55.58 | 150.69 | 1.24 | 1.33 | 24.34 | 261.89 | 0.54 | 2.32 |
| 50 | 74.3 | 64.14 | 144.34 | 1.43 | 1.28 | 23.65 | 361.03 | 0.53 | 3.19 |
| 60 | 74.3 | 69.64 | 165.42 | 1.55 | 1.46 | 23.60 | 443.80 | 0.53 | 3.92 |
| 70 | 74.3 | 72.69 | 177.06 | 1.62 | 1.57 | 23.92 | 519.76 | 0.53 | 4.60 |
| 80 | 74.3 | 74.78 | 182.86 | 1.67 | 1.62 | 24.19 | 546.47 | 0.54 | 4.83 |
| 90 | 74.3 | 75.27 | 189.28 | 1.68 | 1.67 | 24.31 | 562.62 | 0.54 | 4.97 |
| Nom | 84.3 | 22.40 | 42.88 | 0.50 | 0.50 | 22.40 | 42.88 | 0.50 | 0.50 |
| 30 | 84.3 | 70.89 | 141.67 | 1.58 | 1.65 | 27.26 | 131.21 | 0.61 | 1.53 |
| 40 | 84.3 | 58.48 | 128.51 | 1.31 | 1.50 | 27.58 | 207.13 | 0.62 | 2.42 |
| 50 | 84.3 | 69.58 | 125.88 | 1.55 | 1.47 | 27.03 | 290.35 | 0.60 | 3.39 |
| 60 | 84.3 | 76.53 | 130.64 | 1.71 | 1.52 | 28.29 | 355.94 | 0.63 | 4.15 |
| 70 | 84.3 | 80.97 | 137.71 | 1.81 | 1.61 | 29.40 | 413.98 | 0.66 | 4.83 |
| 80 | 84.3 | 83.11 | 144.08 | 1.86 | 1.68 | 29.85 | 441.33 | 0.67 | 5.15 |
| 90 | 84.3 | 84.08 | 153.43 | 1.88 | 1.79 | 30.11 | 449.93 | 0.67 | 5.25 |
| Nom | 94.3 | 22.40 | 33.62 | 0.50 | 0.50 | 22.40 | 33.62 | 0.50 | 0.50 |
| 30 | 94.3 | 67.29 | 139.44 | 1.50 | 2.07 | 33.49 | 105.55 | 0.75 | 1.57 |
| 40 | 94.3 | 59.33 | 109.12 | 1.32 | 1.62 | 33.31 | 160.47 | 0.74 | 2.39 |

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| 50 | 94.3 | 74.27 | 119.76 | 1.66 | 1.78 | 34.07 | 222.25 | 0.76 | 3.31 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 60 | 94.3 | 84.17 | 129.27 | 1.88 | 1.92 | 34.44 | 279.71 | 0.77 | 4.16 |
| 70 | 94.3 | 90.12 | 134.20 | 2.01 | 2.00 | 35.88 | 314.87 | 0.80 | 4.68 |
| 80 | 94.3 | 93.16 | 135.12 | 2.08 | 2.01 | 36.57 | 342.56 | 0.82 | 5.10 |
| 90 | 94.3 | 94.47 | 131.93 | 2.11 | 1.96 | 36.94 | 364.84 | 0.82 | 5.43 |
| Nom | 104.3 | 22.40 | 27.06 | 0.50 | 0.50 | 22.40 | 27.06 | 0.50 | 0.50 |
| 30 | 104.3 | 70.02 | 124.99 | 1.56 | 2.31 | 41.28 | 88.58 | 0.92 | 1.64 |
| 40 | 104.3 | 66.46 | 102.93 | 1.48 | 1.90 | 42.22 | 129.26 | 0.94 | 2.39 |
| 50 | 104.3 | 77.58 | 118.71 | 1.73 | 2.19 | 42.45 | 167.36 | 0.95 | 3.09 |
| 60 | 104.3 | 86.10 | 130.55 | 1.92 | 2.41 | 41.64 | 213.60 | 0.93 | 3.95 |
| 70 | 104.3 | 94.97 | 138.89 | 2.12 | 2.57 | 43.55 | 244.88 | 0.97 | 4.53 |
| 80 | 104.3 | 99.87 | 140.17 | 2.23 | 2.59 | 44.83 | 258.53 | 1.00 | 4.78 |
| 90 | 104.3 | 101.68 | 138.04 | 2.27 | 2.55 | 45.23 | 283.25 | 1.01 | 5.23 |
| Nom | 114.3 | 22.40 | 22.24 | 0.50 | 0.50 | 22.40 | 22.24 | 0.50 | 0.50 |
| 30 | 114.3 | 73.53 | 117.00 | 1.64 | 2.63 | 49.53 | 75.22 | 1.11 | 1.69 |
| 40 | 114.3 | 73.20 | 101.54 | 1.63 | 2.28 | 53.49 | 110.99 | 1.19 | 2.49 |
| 50 | 114.3 | 85.25 | 113.21 | 1.90 | 2.54 | 56.21 | 139.41 | 1.25 | 3.13 |
| 60 | 114.3 | 93.11 | 126.35 | 2.08 | 2.84 | 56.04 | 168.15 | 1.25 | 3.78 |
| 70 | 114.3 | 97.29 | 135.36 | 2.17 | 3.04 | 52.93 | 190.44 | 1.18 | 4.28 |
| 80 | 114.3 | 95.82 | 139.75 | 2.14 | 3.14 | 52.05 | 216.36 | 1.16 | 4.86 |
| 90 | 114.3 | 95.39 | 136.35 | 2.13 | 3.06 | 53.42 | 218.90 | 1.19 | 4.92 |
|  |  |  |  |  |  |  |  |  |  |




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## ASME B31.3 SIF

The in-plane SIF calculated with ASME B31.3 Appendix D is 3.39 and the out-of-plane SIF is 4.18 .

## Error sources

This analysis was ONLY performed to get a rough estimate. Potential error sources are mesh and meshing settings, singularities, load settings, the 3D model, etc.

## Discussion/Conclusion

The in-plane ASME B31.3 SIF (3.39) and the FEA SIF (3.14) are quite close in values.
The out-of-plane ASME B31.3 SIF (4.18) and the FEA SIF (5.43) which is a larger difference. Some of this difference could be that the crotch thickness in my model $(7.1 \mathrm{~mm}$ ) is slightly less than what's required by the code $(7.5 \mathrm{~mm})$. Other errors might be singularities.

The maximum SIFs are located at about 80 degrees of angle. The reason for this is most likely the non-symmetric boundary conditions. Anyhow the SIFs at 80 degrees are not very elevated compared to 90 degrees.

The origin of the LinkedIN discussion focused on whether a 45 degree branch should have an elevated SIF or not. The results from this simple analysis shows that a 45 degree branch have a much lower in-plane and out-of-plane SIFs than the 90degree branch.

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## Referances

/1/ Paulin Reseach Group - http://www.paulin.com/WEB_Markl_SIFs_ASME_VIII_2.aspx


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